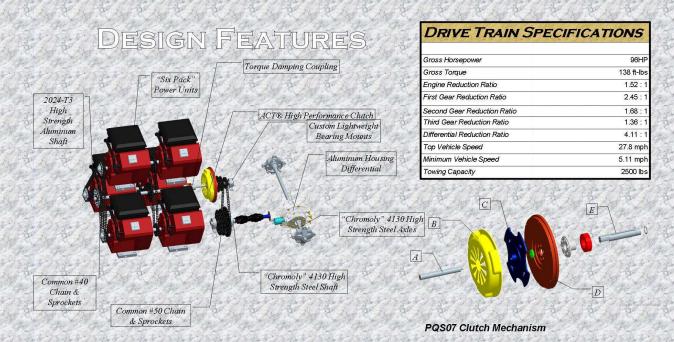
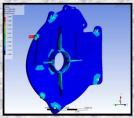
DRIVE TRAIN DESIGN FOR PQS07

Designer: Greg Long







Stress analysis of clutch bearing plate that supports the thrust loading of the clutch. Gusset bracing was added to reduce stresses from FEA analysis

DRIVE TRAIN BILL OF MATERIALS

d		
	Components	Material
	Engine Sprockets	#40 Chain Steel Type
Š	Common Shaft Sprockets	#40 Chain Steel Type
	Chain for Engines	Common #40 Chain
ij	Common Shaft	2024-T3 Aluminum
	Flange Bearings	1" Roller Bearing
98	LoveJoy Coupler	L100 Series Cast Iron
Ŋ	Key Stock	1/4" and 3/8" Mild Steel
d	Geo Metro Flywheel	10 Cast Iron
Š	ACT Clutch	7 1/2" High Performance
38	Timken TSL Bearings	Sealed Tapered Roller
2	Transmission Shaft (upper)	1" 4130 High Strength Steel
ĝ	Transmission Sprockets	#50 Chain Steel Type
É	Transmission Chain	Common #50 Chain
Š	Transmission Shaft (lower)	1 1/4" 4130 High Strength Steel
ß	Bottom Shaft Bearings	Sealed Roller
à	Polaris Driveshaft	Premium Cast Iron
	Dana Differential	Aluminum Housing
Š	Polaris Axles	Case Hardened Steel
3	Axle Shaft Material	1 1/4" 4130 High Strength Steel
à	Axle Tubes	2 1/2" Mild Steel Tubing
	Axle Bearings	40mm Polaris Roller
Š	Rear Wheel Hubs	Premium Cast Aluminum

Design for Strength

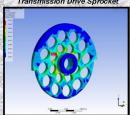
- Verify component dimensions based on predicted stresses of the particular component
- Efficient allocation of material to optimize the power per weight ratio of the tractor

drive train.

Design for Weight

- Stress analysis determines how much unneeded material can be deleted from a component
- Reducing drive train weight reduced rotating inertia and increases power transfer efficiency

Transmission Drive Sprocket



Drive sprockets were analyzed to determine how much unneeded material could be removed. FEA was conducted to optimize the power to weight ratio of the drive train

Stress analysis was conducted on the predicted high stress components of the drive train. Finite element analysis (FEA) was conducted through the use of ANSYS Workbench® to provide verification of design stresses. Components such as shafts, sprockets, mounting plates, and axles were analyzed with FEA to optimized the design for both strength and weight.

Custom fabrication of components provided valuable shop experience while learning the basics of machining and welding. Most components are stock items that required minor modifications.

Common sizes were used to make the components of the tractor readily replaceable and serviceable. Modifications included keyway cutting, axle shortening, flywheel, and sprocket weight reduction.

Newly designed components were the custom aluminum bearing mounts and the flywheel mounting hub.

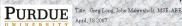
Torque transfers from common shaft [A] to friction disk [C]. Engagement and disengagement occurs by means of operating the pressure plate [B] causing the pressure plate and flywheel [D] to rotate with the friction disk. Torque is then transferred to the output shaft [E] to power the rest of the



Front drive sprockets and chains where manufactured for lightweigh and strength. Engine configuration tests proved the drive train



Reduction of weight was critical in the design of the drive train. Each sprocket was machined to optimize the power transfer to weight ratio.



QUARTER SCALE TRACTOR DESIGN COMPETITION

Designers: Greg Long—Drive-Train John Mahrenholz—Frame April 18, 2007 Acknowledgements

Academic Support:

Agricultural and Biological Engineering Dept

- Organizational Support:

 American Society of Agricultural Biological
 Engineering Competition leadership and
 organization

 National Fluid Power Association –
 Donations to purchase materials to build
 hydraulic dynamometer to test and brake-in
 engines

- Industry Support:

 † Daimler Chrysler Grant support

 † AED Motorsports Discount raw materials

 † Polaris Industry Discount drive-train
- components

 Dana-Spicer Donation of differential

 Real Wheel Discount on rear rims
- ♦ Real Wheer Location
 ♦ BMF Machining
 ♦ Kaman Ind. Discount drive-train materials
 Competition Sponsors Judge and

Advisors: Academic Advisor – Dr. Joseph Irudayaraj Technical Advisor - Dr. John Lumkes

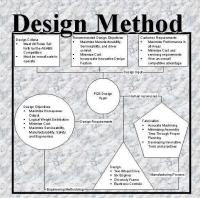






DETAILED WEIGHT ANALYSIS

Subsystem	Part or Assy	Weight (lbs)	Subsystem	Part or Assy	Weight (lbs)
Drive-Train	Engine Sprokets (6)	5.34	Frame	Back Main Frame Rails	9.87
Drive-Train	Com. Shft. Sprokets (2)	3.8	Frame.	Front Lower Frame Rails	3.38
Drive-Train	Chain (40)	5	Frame	Middle Brace	2.84
Drive-Train	Com. Shft.	2.1	Frame	Front Bolster Mount	0.39
Drive-Train	Com. Shft. Bearings (2)	0.75	Frame	Front Axle	14.84
Drive-Train	LoveJoy Coupler	1.25	Frame	Lower Engine Mounts	5.74
Drive-Train	Clutch Input Shft.	0.5	Frame	Back Top Frame Rails	1.97
Drive-Train	Bellhousing w/ bearings	8.5	Frame	Back Angle Frame Rails	2.08
Drive-Train	Bellhousing plate assembly	3.125	Frame	Bell Housing Mount	4.17
Drive-Train	Clutch/Flywheel	16	Frame	Frame Cross Member	1.24
Drive-Train	Starter (Reverse)	6	Frame	Large Gussets	0.34
Drive-Train	Clutch Output Shft	1.34	Frame	Back Up Angle Frame Rails	1.68
Drive-Train	50_16 Sprok	0.8	Frame	Right Rear Axle Tube	7.91
Drive-Train	50_21	1.12	Frame	Left Rear Axle Tube	4.18
Drive-Train	50_23	1.25	Frame	Hitch Assembly	5.22
Drive-Train	Clutch Out Shft Pillow w/bear	0.6	Frame	Vertical Chain Support	0.64
Drive-Train	Chain (50)	1.26	Frame	Wheelie Bar Supports	2.04
Drive-Train	Bottom Shft	2.8	Frame	Bumper Bars	0.87
Drive-Train	Bottom Shft Pillow w/bear (2)	2	Frame	Rear Axle Mounts	2.69
Drive-Train	50_43	4.68	Frame	Back Up Angle Flanges	0.24
Drive-Train	50_38	3.15	Frame	Back Up Angle Flanges2	0.19
Drive-Train	50_36	2.35	Frame	Chain Bank Shields	3.61
Drive-Train	Driveshaft	4.75	Frame	Chain Brace	0.07
Drive-Train	Dana Diff	35	Frame	Rack Brace	1.88
Drive-Train	Axles	8	Frame	Engine Chain Shields	10.41
Drive-Train	Rear Tires	60	Frame	Chain Guides	0.39
Drive-Train	Rear Hubs/bearings	5	Frame	Engine Triangle Mount	5.05
Drive-Train	Engines (6)	480	Frame	Under Top Engine Support	2.03
	Subtotal	666.465	Frame	Under Top Engine Supp2	1.63
			Frame	Front Upper Frame Rails	3.11
Drive-Train	Subtotal	666.465	Frame.	Small Engine Spacers	0.33
Frame	Subtotal	123.38	Frame	Large Engine Spacers	0.38
Steering	Front End Assembly	17.2	Frame	Engine Triangle Vert. Brace	1.97
Electronics	Everything	35	Frame	Bolts	10
Driver's Station	Carbon, Pedals, Floor Pans	14	Frame	Welds	10
Fuel	Everything	7		Subtotal	123.38
Brakes	Calipers, Rotors, Lines	4			
Exhaust	Everything	20			
Total	1 7	887			

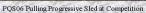


Frame		
	11-1-0	
Components	List Cost	Our Cost
1.00 x .125 6061 T6 DRAWN RD TUBE ALUM		\$30.90
1 X 1 X .065 4130 SQUARE TUBE	\$440.02	440.02
1.00 X 1.00 X .125 HR STEEL ANGLE	\$14.83	\$14.83
3"x 25"x20" Aluminum Plate 6061 T6 Aluminum I-beam	\$200.00	\$200.00 \$30.00
	\$30.00	7/
Sheet-metal	\$150.00	\$0.00
Hitch Material	\$100.00	\$10.00
Sub-Total	\$965.75	\$725.75
Drive-Train		
Components	List Cost	Our Cost
Engine Sprockets	\$49.08	\$49.08
Common Shaft Sprockets	\$22.86	\$22.86
Chain for Engines	\$29.20	\$29.20
Common Shaft	\$57.82	\$57.82
Flange Bearings	\$10.00	\$10.00
LoveJoy Coupler	\$26.00	\$26.00
Key Stock	\$4.00	\$0.00
Geo Metro Transmission	\$1,200.00	\$150.00
ACT Clutch	\$267.00	\$0.00
Timken TSL Bearings	\$40.00	\$40.00
Transmission Shaft (upper)	\$14.03	\$14.03
Transmission Sprockets	\$76.07	\$76.07
Transmission Chain	\$21.00	\$21.00
Transmission Shaft (lower)	\$21.94	\$21.94
Bottom Shaft Bearings	\$48.57	\$48.57
Polaris Driveshaft	\$98.79	\$50.00
Dana Differential	\$540.00	\$0.00
Polaris Axles	\$561.58	\$110.00
Axle Shaft Material	\$4.00	\$0.00
Axle Tubes	\$520.00	\$130.00
Axle Bearings	\$3.12	\$2.00
Rear Wheel Hubs	\$93.59	\$16.00
Snap Rings (External and Internal)	\$17.85	\$17.85
Chain Rub Blocks for Chain Idler	\$11.58	\$11.58
Bell housing Plate	\$5.00	\$0.00
Top Shaft Bearing Housing	\$7.00	\$0.00
Axle Tube Bearing Mount Material	\$6.00	\$0.00
Sub-Total	\$3,756.08	\$904.00
Labor and Engineering Time		
	(per hr)	Cost
Skill Level A 50	\$40.00	\$2,000.00
Skill Level B 70	\$20.00	\$1,400.00

DESCRIPTION OF THE COMPETITION

The ASABE International Quarter Scale Competition challenges students to design and build a small scale pulling tractor to compete against other colleges. Through involvement in the competition, students gain practical experience in the design of drive systems structural frame, tractor performance, and material characteristics. In addition, students develop skills in communication, leadership, teamwork, fundraising, and test & development. The competition prepares students to be future professional engineers.

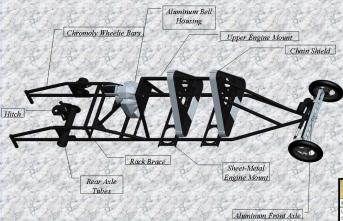




FRAME DESIGN FOR PQS07

Designer: John Mahrenholz

DESIGN FEATURES



Details

•Material-ANSI 4130 Steel (a.k.a. Chromoly)

High Strength: Density Ratio

Easily welded

Low carbon steel can also be

welded to reduce costs

• Cross Section—Square Tube

Easy to cut

Use Engines for Structure Better Visibility

Class	1050	1300	Units
Max Chain Load Predicted	1900	2500	lbf
Min Chain Load Predicted	1250	1500	lbf
Chain Angle	20.2434	20.2434	deg
Max Dynamic Weight Transfer from Chain	657.4169	865.0223	lbf
Min Dynamic Weight Transfer from Chain	432.5111	519.0134	lbf
Tire Roling Radius	12	12	in
Max Dynamic Torque Transfer from Load	21391.67	28146.94	in*lbf
Min Dynamic Torque Transfer from Load	14073.47	16888.16	in*lbf
Horizontal Distance from Hitch Pt. to Axle	5	5	in
Max Dynamic Torque Transfer from Chain	3287.085	4325.111	in*lbf
Min Dynamic Torque Transfer from Chain	2162.556	2595.067	in*lbf
Max Total Dynamic Torque on Axle	24678.76	32472.05	in*lbf
Min Total Dynamic Torque on Axle	16236.02	19483.23	in*lbf
No Ballast Tractor Weight	1040	1040	lbf

in Total Weight on Front @ End of Pull



The Completed rolling chassis of the frame weighs only 225lb

Class	1050	1300	Units
Max Chain Load Predicted	1900	2500	lbf
Min Chain Load Predicted	1250	1500	lbf
Chain Angle	20.2434	20.2434	deg
Max Dynamic Weight Transfer from Chain	657.4169	865.0223	lbf
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Min Total Dynamic Torque on Axle	16236.02	19483.23	in*lbf
No Ballast Tractor Weight	1040	1040	lbf
Estimated No Ballast Tractor CG	27.3	27.3	in
Moment Created from CG	28392	28392	in*lbf
Wheel Base	87	87	in
Available Ballast Weight	10	260	lbf
Ballast Location from Rear Axle	-10	35	in
Min Total Mainht on Front @ Find of Bull	9.740051	7.000070	Hof

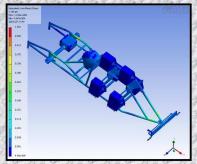
Easy to jig Shape—Expanded Truss

Manufacturability	Safety	Ergonomics	Serviceability
Use of steel as main component decreases weld complications Limited number of welded joints Expanded cross section eases component placement	Extra enforcement around the operator station limits the risk of operator injury due to frame failure. Sharp corners on the frame have been eliminated to reduce the risk of operator and service technician injury.	The expanded cross section allows engines to be place lower which increases operator visibility. Free rotating bolster style front end increases ride comfort over rough terrain	Steel components allow most welds to be repaired at local machine shops by an experienced welder. The expanded cross section allows easy engine and drive-train removal for servicing the tractor.

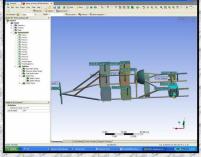
odeling & Analysis

Pro/Engineer® for Solid Modeling

- Determine spatial locations for components Determine how much material to remove
- Easy modification without cost of material . Under given loads, will the frame fail?
- ANSYS Workbench® for Stress Analysis

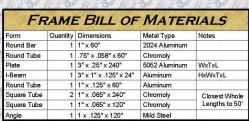


Along with analysis of individual components, the entire frame was analyzed to determine the effects of combined loading



Loading conditions were determined from analysis of previous pulling performance, traction estimates and drive-train gearing ratios.

FABRICATION





ndividual Frame Rails were place on the jig table ecured with wooden blocks. This method insure ails would be identical once taken out of the jig.

